IEE Solar Combi⁺ WP3 – Virtual Case Studies

Actual project stage

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Subtasks of WP3

- **3.1** Preparation of system implementation in simulation tools
- **3.2** Definition of applications (3-5) and locations to be studied
- **3.3** Determination of loads for the applications and locations
- 3.4 Determination of basic system configurations and control strategies
- 3.5 Simulation study (variation of locations, buildings, components and parameters)
- **3.6** Energy-related evaluation of case studies
- **3.7** Economic-related evaluation of case studies



Locations

- I. Office building cold distribution system: fan coils, supply air cooling (10°C/15°C)
- II. Residential building cold distribution system: fan coils (10°C/15°C)
- III. Residential building cold distribution system: chilled ceilings, etc. (15°C/18°C)



Ecoheatcool: European heating index EHI

3 climatic zones

Selected by EHI / ECI:

- 100 / 100 Strasbourg, Franc
- 85 / 115 Toulouse, France
- 70 / 140 Naples, Italy

Areas with similar EHI / ECI index (latent loads not considered):

🔵 100 / 100

70 / 140



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Source: Ecoheatcool / WP1: European heat market

Ecoheatcool: European cooling index ECI

3 climatic zones

Selected by EHI / ECI:

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Areas (latent loads not considered) with respective index EHI / ECI index:

🔵 100 / 100

70 / 140



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Source: Ecoheatcool / WP2: European cold market

- Building standard:
- not according to assumption in ECOHEATCOOL:

insulation ~ [heat-degree-days]^{1/2}

Unchanged (according to available reference building models)





- I. Office building cold distribution system: fan coils, supply air cooling (10°C/15°C)
- → One-storied office building
- → Ca. 310m² cooled floor area
- Based on Task 38 reference building





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Load profile of office building for Toulouse

 I. Office building cold distribution system: fan coils, supply air cooling (10°C/15°C)

OFFICE (310 m ² conditioned area)										
	Heating			Cooling				DHW		
	sensible	latent	sensible	latent	sensible	latent	sensible	latent	-	-
	kWh/a	kWh/a	kWh/	kWh/	kWh/a	kWh/a	kWh/	kWh/	kWh/a	kWh/
			m²*a	m²*a			m²*a	m²*a		m²*a
Strasbourg	20999	621	67.74	2.00	10303	295	33.24	0.95	0	0
Toulouse	10437	143	33.67	0.46	15102	484	48.72	1.56	0	0
Naples	2796	62	9.02	0.20	22537	2496	72.70	8.05	0	0

- → Heating load: sensible load
- → Cooling load: sensible + latent load









- II / III. Residential buildings
 II. cold distribution system: fan coils (10°C/15°C)
 III. cold distribution system: chilled ceiling (15°C/18°C)
- → Two-storied building
- → 140 m² cooled floor area
- → Based on Task 32 reference buildings



- → 2 building standards (60kWh/m² and 100kWh/m² in Zurich, Switzerland)
- → Only 2 locations (Toulouse and Naples)



- II / III. Residential buildings
 - II. cold distribution system: fan coils (10°C/15°C)

III. cold distribution system: chilled ceiling (15°C/18°C)

Residential (140 m ² conditioned area)										
	Heating			Cooling				DHW		
	sensible	latent	sensible	latent	sensible	latent	sensible	latent	-	-
	kWh/a	kWh/a	kWh/	kWh/	kWh/a	kWh/a	kWh/	kWh/	kWh/a	kWh/
			m²*a	m²*a			m²*a	m²*a		m²*a
Strasbourg *	-	-	-	-	-	-	-	-	-	-
Toulouse 60	3361	82	24.00	0.58	800	58	5.71	0.41	1772	12.66
Toulouse 100	6376	71	45.54	0.51	726	71	5.18	0.51	1772	12.66
Naples 60	1265	46	9.04	0.33	2032	412	14.51	2.94	1600	11.43
Naples 100	2917	39	20.83	0.28	2135	449	15.25	3.20	1600	11.43

Fan coils

- → Heating load: sensible load
- → Cooling load: sensible + latent load

Chilled ceiling:

- → Heating load: sensible load
- → Cooling load: sensible load



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3.3: Determination of loads

Problem: Building models with fixed geometry

chillers with 4.5 kW – 15 kW nominal chilling capacity + fixed load files \rightarrow difficult to compare

Approach

 \rightarrow scaling of load file

 $f_{scale} = \frac{Q_{chill,ref}}{Q_{coolingload,max}} \qquad \begin{array}{l} \mathsf{Q}_{chill,ref}: \ \mathsf{Chilling power at reference} \\ \ \mathsf{temperatures} \\ \mathsf{Q}_{coolingload,max}: \ \mathsf{Maximum cooling load} \quad \mathsf{of} \\ \ \mathsf{load file} \end{array}$



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3.3: Determination of loads

Chilling power at reference inlet temperatures

- hot water circuit: 80°C for flat plate collector
 85°C for evacuated tube collector
- cooling water circuit: 27°C for wet cooling tower
 - 35°C for dry cooling tower
- chilled water circuit: 18°C for chilled ceiling 15°C for fan coil
- Maximum cooling load of load file
 - maximum sensible + latent cooling load for fan coil
 - maximum sensible cooling load for chilled ceiling



3.4: Basic system configurations

- 2 standard configurations have been defined
- 1 configuration does NOT meet "Spanish requirement"
- Each chiller will be simulated with one configuration
- Variation of components
 - collector (flat plate or vacuum tube)
 - cooling tower (wet or dry)
 - cold distribution system (fan coil or chilled ceiling)
 - load file (building and location)



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3.4: System configuration C1

Simulated for:



3.4: System configuration E1

Simulated for:



3.4: Collectors

Standard Trnsys components



Collector	Туре	Tested flow rate (kg/hrm²)	η₀	a ₁ (W/ Km²)	a ₂ (W/ K²m²)
SolTop Cobra X	Flat plate	120	0.823	3.02	0.0125
Phönix CPC 14/21	Evacuated tube	30	0.601	0.767	0.0038

Data referring to aperture area



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3.4: Variation of collector area and storage volume

- Collector (aperture area):
 - 5 steps
 - 2 ... 5 m²/kW reference chilling capacity
 - Hot water storage (volume):
 - 3 steps
 - 25 50 75 l/m² aperture area



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3.4: Chiller models

- ClimateWell: physical model provided by ClimateWell; 2*Type 215 (barrels) + Type 216 (Controller)
- EAW: Type 307 (adapted Type 107) by Eurac
- Rotartica: characteristic model by ISE; Type 231
- Sonnenklima: because of problems with Type 177 characteristic model by ISE; Type 233
- Sortech: model provided by Sortech, implemented in Fortran by ISE; Type 290



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3.4: Heat rejection

- Dry cooler
 - model developed by Francesco Besana (Eurac)
 - definition of a *standard cell* with one fan
 - number of cells can be selected by user (adaption to chiller)



3.4: Heat rejection

- Wet cooling tower
 - Trnsys standard Type 51
 - parameters ???
 - Sonnenklima
 - Axima
 - Type 551 by TU Berlin ??

				Type51h
				Typesie
T	ype	e51b		
	Inpu	it Output Derivative Special Cards	External Files Co	mment
T	đ	Number of tower cells	1	-
Ĩ	đ	Maximum cell flow rate	40.0	m^3/hr
Ĩ	đ	Fan power at maximum flow	1.0	kw I
Ĩ	đ	Minimum cell flow rate	10.0	m^3/hr
Ĩ	đ	Sump volume	1.0	m^3
Ĩ	đ	Initial sump temperature	15.0	c I
I	đ	Mass transfer constant	2.3	-
Ī	£	Mass transfer exponent	-0.72	



3.4: Control strategy

- Collector circuit
 - Δ T-controlled
 - Flow rate dependent on irradiation :

150 W/m² ... 800 W/m² corresponds to 20% ... 100 %

- Boiler
 - System C1: if solar heat insufficient then 80°C for chiller; 40°C for heating; 60°C for DHW
 - System E1:

winter: storage top 200 I maintained at 60°C

summer: storage top 200I maintained at 75°C (day) / 70°C (night)



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3.4: Control strategy

- Chiller
 - runs if cooling demand
 - 1 kWh storage implemented (integrator)
 - \rightarrow to prevent clocking at low cooling loads
 - \rightarrow to reproduce inertia of building
- Control of chilling capacity
 - by fan of heat rejection unit (Sonnenklima)
 - internally by duration of cycles (Sortech)
 - by hot water temperature (Rotartica)
 - none \rightarrow cold water storage (EAW)
 - ClimateWell ??



3.5: Simulation study

Division of simulation work:

System Chiller	C1	E1
ClimateWell	EURAC	
EAW		AEE Intec
Rotartica	UNIBG	
Sonnenklima	CRES	
Sortech		ISE



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3.5: Simulation study



ISE

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Chilled Ceiling

3.5: Simulation study

- Strasbourg: 1*2*2*3*5 = 60 variations
- Toulouse: 5*2*2*3*5 = 300 variations
- Naples: 5*2*2*3*5 = 300 variations
- Sub-total: 660 variations per chiller
- TOTAL: 3300 variations FanCoil Residential 60 5 Collector Chilled Storage 1 sizes Ceilina Flat-Plate Naples Wet Cooling 5 Collector Storage 2 Strasbourg Office FanCoil Tower sizes Evacuated Toulouse Tube 5 Collector Storage 2 FanCoil sizes Dry Cooling Residential Tower 100



3.1: Simulation outputs – annual values *

- Thermal energies:
 - sources: collector, boiler
 - consumers: DHW, heating, cooling
 - chiller: all 3 circuits
 - Electric energies:
 - heat sources: collector, boiler
 - chiller: all 3 circuits; heat rejection
- Reference system
 - compression chiller
 - boiler
- Rest: water consumption, …



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3.1: Simulation output data

Toulouse, office, dry cooling, flat plate collector, Sortech





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3.6, 3.7: energetic and economic evaluation

- **Performance figures of** the system: Collector efficiency, collector yield, solar fractions, COP, ...
- **Environmental** performance figures: PEsavings, PE-COP, CO2savings, PER, ...
- **Economical figures:** Investment costs, annual costs, costs per saved kWh PE....

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Next steps

- Definition of performance figures
- Completion of dry cooling tower
- Decision on wet cooling tower
- Testing of stability of Trnsys decks
- Carrying out simulations
- Verification of results

. . .

Comparison of system C1 and E1



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Deliverables

D3.1

Database with case studies: description and results month $13 \rightarrow \text{month } 17$

D3.2

Report with description of methodology month $13 \rightarrow \text{month } 17$

D3.3

Report on results month $13 \rightarrow \text{month } 17$



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